

# Phase Angle

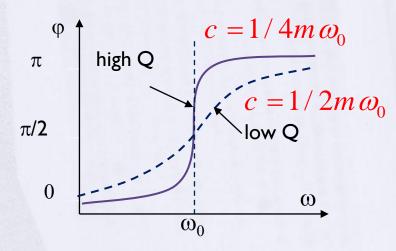
$$tg\,\varphi = \frac{\omega c}{k - m\,\omega^2}$$

$$\gamma = c / 2m$$

$$tg\,\varphi = \frac{2\gamma\omega}{\omega_0^2 - \omega^2}$$

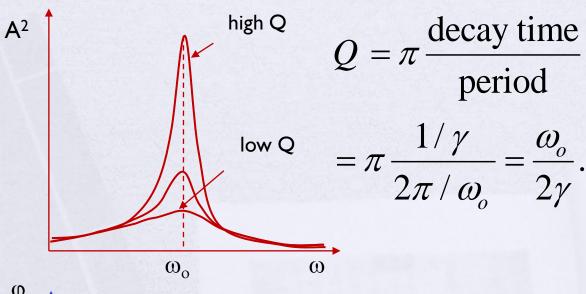
$$\varphi = tg^{-1} \frac{2\gamma\omega}{\omega_0^2 - \omega^2}$$

$$\omega_0 = (k / m)^{1/2}$$

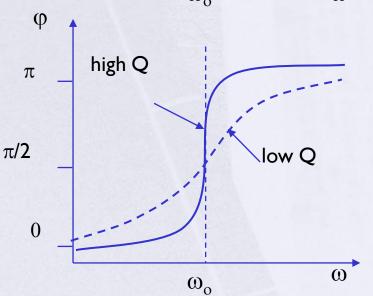


High Q = sharp resonance Damping reduces Q

The *width* of the resonance curves depends on c, i.e., on the amount of damping



Bandwidth, BW = difference between the two half-power frequencies



The higher the Q, the smaller the bandwidth

#### Velocity Resonance

$$x = A \cos(\omega t - \varphi)$$
  $\dot{x} = -\omega A \sin(\omega t - \varphi)$ 

$$v(\omega) = \frac{\omega F_0 / m}{\sqrt{(\omega_0^2 - \omega^2) + 4\gamma^2 \omega_0^2}} \qquad \gamma > \frac{\omega_0}{\sqrt{2}}$$

Example: The exponential damping factor  $\gamma$  of a spring suspension system is one-tenth the critical value. If the undamped frequency is  $\omega_0$ , find (a) the resonant frequency, (b) the quality factor, (c) the phase angle  $\varphi$  when the system is driven at a frequency  $\omega = \omega_0/2$ , and (d)the steady-state amplitude at this frequency.

$$a)\gamma = \gamma_{crit} / 10 = \omega_0 / 10 \qquad \omega_r^2 = \omega_d^2 - \gamma^2$$

$$\omega_r = \sqrt{\omega_0^2 - 2(\omega_0 / 10)^2} = \omega_0 \sqrt{0.98} = 0.99\omega_0$$

b)Q = 
$$\frac{\omega_0}{2\gamma} = \frac{\omega_0}{2(\omega_0/10)} = 5$$

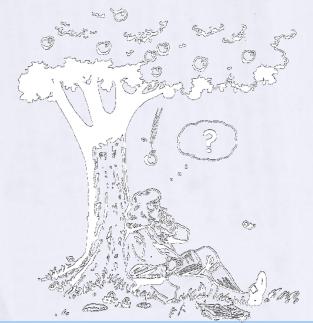
c)
$$\varphi = tg^{-1} \frac{2\gamma\omega}{\omega_0^2 - \omega^2} = tan^{-1} \left[ \frac{2(\omega_0/10)(\omega_0/2)}{\omega_0^2 - (\omega_0/2)^2} \right] = 7.6^{\circ}$$

$$d A(\omega) = \frac{F_0 / m}{\left[ (\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2 \right]^{1/2}} \qquad \omega = \omega_0 / 2$$

$$A(\omega) = \frac{F_0/m}{\left[ (\omega_0^2 - \omega_0^2/4)^2 + 4(\omega_0/10)^2(\omega_0/2)^2 \right]^{1/2}}$$

$$A(\omega) = \frac{F_0 / m}{0.7506\omega_0^2} = 1.332 \frac{F_0}{m \omega_0^2}$$

$$\frac{F_0}{m\,\omega_0^2} = \frac{F_0}{k}$$



#### The Nonlinear Oscillator. Method of Successive Approximations

$$F(x) = -kx$$
 Linear Oscillator
$$F(x) = -kx + \varepsilon(x)$$
 Nonlinear Oscillator
$$\varepsilon(x) = \varepsilon_2 x^2 + \varepsilon_3 x^3 + \dots$$

$$m\ddot{x} + kx = \varepsilon_2 x^2 + \varepsilon_3 x^3 + \dots$$

$$m\ddot{x} + kx = \varepsilon_2 x^2 + \varepsilon_3 x^3 + \dots$$

$$\ddot{x} + \frac{k}{m} x = \frac{\varepsilon_3}{m} x^3$$

$$\ddot{x} + \omega_0^2 x = \lambda x^3$$

We shall find the solution by the method of successive approximations.

$$\lambda = 0 \rightarrow x = A \cos \omega_0 t$$

Suppose we try a first approximation of the same form

$$\lambda \neq 0 \rightarrow x = A \cos \omega t \qquad \omega \neq \omega_0$$

$$-A \omega^2 \cos \omega t + A \omega_0^2 \cos \omega t = \lambda A^3 \cos^3 \omega t$$

$$= \lambda A^3 \left(\frac{3}{4} \cos \omega t + \frac{1}{4} \cos 3\omega t\right)$$

$$\cos 3\alpha = 4 \cos^3 \alpha - 3 \cos \alpha$$

$$(-\omega^2 + \omega_0^2 - \frac{3}{4} \lambda A^2) A \cos \omega t - \frac{1}{4} \lambda A^3 \cos 3\omega t = 0$$

$$-\omega^{2} + \omega_{0}^{2} - \frac{3}{4}\lambda A^{2} = 0 \qquad \omega^{2} = \omega_{0}^{2} - \frac{3}{4}\lambda A^{2}$$

$$\omega = f(A) \qquad \omega = \sqrt{\frac{k}{m}}$$

 $x = A \cos \omega t + B \cos 3\omega t$  Second trail solution

$$(-\omega^{2} + \omega_{0}^{2} - \frac{3}{4}\lambda A^{2})A\cos\omega t + (-9B\omega^{2} + \omega_{0}^{2}B - \frac{1}{4}\lambda A^{3})\cos3\omega t + (terms\ involving\ B\lambda\ and\ higher\ multiples\ of\ \omega t) = 0$$

$$-9B\omega^{2} + \omega_{0}^{2}B - \frac{1}{4}\lambda A^{3} = 0 \qquad B = \frac{\frac{1}{4}\lambda A^{3}}{-9\omega^{2} + \omega_{0}^{2}}$$

$$B = \frac{\frac{1}{4}\lambda A^{3}}{-9\omega^{2} + \omega_{0}^{2}} = \frac{\lambda A^{3}}{-32\omega_{0}^{2} + 27\lambda A^{2}} \simeq -\frac{\lambda A^{3}}{32\omega_{0}^{2}}$$
$$x = A\cos\omega t - \frac{\lambda A^{3}}{32\omega_{0}^{2}}\cos 3\omega t$$
$$\omega = \omega_{0}(1 - \frac{3\lambda A^{2}}{4\omega_{0}^{2}})^{1/2}$$

### fxample: The simple pendulum as a nonlinear oscillator

$$\ddot{\theta} + (g/l)\sin\theta = 0$$

$$\sin\theta \simeq \theta$$

$$\sin \theta = \theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!}$$

$$\ddot{\theta} + \frac{g}{l}\theta = \frac{g\theta^3}{3!l}$$

$$\frac{g}{l} = \omega_0^2$$

$$\ddot{\theta} + \omega_0^2 \theta = \frac{\omega_0^2}{3!} \theta^3$$

$$\ddot{\theta} + \omega_0^2 \theta = \lambda \theta^3$$

$$\lambda = \frac{\omega_0^2}{3!} = \frac{\omega_0^2}{6}$$

$$\omega = \omega_0 (1 - \frac{3\lambda A^2}{4\omega_0^2})^{1/2}$$

$$\omega = \omega_0 \left[ 1 - \frac{3(\frac{\omega_0^2}{6})A^2}{4\omega_0^2} \right]^{1/2} = \omega_0 (1 - \frac{A^2}{8})^{1/2}$$

$$T = \frac{2\pi}{\omega} = 2\pi\sqrt{l/g}\left(1 - \frac{A^2}{8}\right)^{-1/2} = T_0\left(1 - \frac{A^2}{8}\right)^{-1/2}$$

$$A = \pi / 2 \text{ rad}$$
  $T = T_0 (1 - \frac{\pi^2}{32})^{-1/2} = 1.2025T_0$ 

#### Chapter 3 Problems

$$x(t) = A \sin(\omega t - \varphi)$$

$$v(t) = A \omega \cos(\omega t - \varphi)$$

The system passes the center at the moment  $t_i$  defined by the condition

$$x(t_i) = A \sin(\omega t_i - \varphi) = 0$$
  $\omega t_i + \varphi = \pi i$ 

where i = 0, 1, 2, 3, ... Velocity at these moments equals

$$v(t_i) = A\omega\cos(\omega t_i - \varphi) = A\omega\cos(\pi i) = \pm A\omega$$

$$\omega = \frac{|v(t_i)|}{A} \qquad T = \frac{2\pi A}{|v(t_i)|}$$

$$T = \frac{2\pi A}{|v(t_i)|}$$

$$T = 2\pi/5 \approx 1.26 \text{ s}$$

3) 
$$f = 10Hz$$
,  $t = 0$ ,  $x_0 = 0.25m$ ,  $v_0 = 0.1m/s$ 

$$x(t) = x_0 \cos \omega t + \frac{v_0}{\omega} \sin \omega t$$

$$x(t) = x_0 \cos(2\pi f t) + \frac{v_0}{2\pi f} \sin(2\pi f t)$$

$$x(t) = 0.25\cos(20\pi t) + \frac{0.005}{\pi}\sin(20\pi t)$$

$$x(t) = A \sin(\omega t - \varphi)$$

$$v(t) = A \omega \cos(\omega t - \varphi)$$

$$v(t) = A \omega \sqrt{1 - \sin^2(\omega t - \varphi)}$$

$$\omega \sqrt{A^2 - A^2 \sin^2(\omega t - \varphi)} = \omega \sqrt{A^2 - x}$$

$$= \omega \sqrt{A^2 - A^2 \sin^2(\omega t - \varphi)} = \omega \sqrt{A^2 - x^2}$$

$$\dot{x}_1 = \omega \sqrt{A^2 - x_1^2} \qquad \dot{x}_2 = \omega \sqrt{A^2 - x_2^2}$$

$$A^2 = \frac{x_1^2 \dot{x}_2^2 - x_2^2 \dot{x}_1^2}{\dot{x}_2^2 - \dot{x}_1^2} \qquad \omega^2 = \frac{\dot{x}_1^2 - \dot{x}_2^2}{x_2^2 - x_1^2}$$

9) 
$$x(t) = A e^{-\gamma t} \cos(\omega_d t - \varphi)$$

The moments  $t_i$  (i = 1, 2, 3, ...) when x(t) has maximums can be determined from the equation

$$\frac{dx}{dt} = -Ae^{-\gamma t_i} \left[ \omega_d \sin(\omega_d t_i - \varphi) + \gamma \cos(\omega_d t_i - \varphi) \right] = 0$$

$$\tan(\omega_d t - \varphi) = -\gamma / \omega_d$$

$$\omega_d t_i + \varphi = \tan^{-1}(-\gamma/\omega_d) + 2\pi i$$

Therefore the time between two successive maxima is

$$t_{i+1} - t_i = \frac{2\pi}{\omega_d}$$

The amplitude of the ith maximum is

$$x(t_i) = A e^{-\gamma t_i} \cos(\omega_d t_i - \varphi)$$

The amplitude of the (i + 1)th maximum is

$$x(t_{i+1}) = Ae^{-\gamma t_{i+1}} \cos(\omega_d t_{i+1} - \varphi)$$
Since  $t_{i+1} = t_i + \frac{2\pi}{\omega_d}$ ,  $x(t_{i+1}) = e^{-\gamma 2\pi/\omega_d} x(t_i)$ 

$$\frac{x(t_{i+1})}{x(t_i)} = e^{-2\pi\gamma/\omega_d}$$
)

the ratio  $x(t_{i+1})/x(t_i)$  does not depend on time and is given by

$$x(t) = e^{-\gamma t} \cos \omega_d t$$

After n cycles the amplitude drop is 1/e, so therefore the time passed is  $T = 1/\gamma$ .

the time of n oscillation is determined by the condition

$$\frac{\omega_d T = 2\pi n}{1} = \frac{2\pi n}{\omega_d} \qquad \omega_d = \sqrt{\omega_0^2 - \gamma^2} \qquad \gamma^2 = \frac{\omega_0^2}{1 + 4\pi^2 n^2}$$

$$\omega_d = \omega_0 \sqrt{1 - \frac{1}{1 + 4\pi^2 n^2}} T_d / T = \omega_0 / \omega_d = \sqrt{1 + \frac{1}{4\pi^2 n^2}}$$

$$A(\omega) = \frac{\gamma A_{\text{max}}}{\sqrt{(\omega_0 - \omega)^2 + \gamma^2}}$$

$$\frac{A}{A_{\text{max}}} = \frac{1}{2}$$

$$\omega_0 - \omega = \sqrt{3}\gamma$$

### Some concepts for oscillations

restoring force:

natural frequency:

undamped oscillations:

damped oscillations:

simple harmonic oscillation:

forced oscillations:

A force causes the system to return to some equilibrium state periodically and repeat the motion

Resonant oscillation period, determined by physics of the system alone. Disturb system to start, then let it go. Examples: pendulum clock, violin string

Idealized case, no energy lost, motion persists forever Example: orbit of electrons in atoms and molecules

Oscillation dies away due to loss of energy, converted to heat or another form. Example: a swing eventually stops

Undamped natural oscillation with F = -kx (Hooke's Law); i.e. restoring force is proportional to the displacement away from the equilibrium state

External periodic force drives the system motion at it's own frequency/period, may not be the resonant frequency

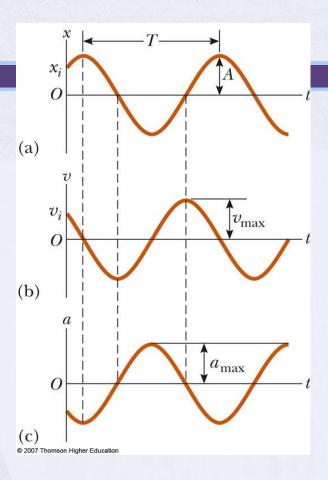
# Forced oscillations and resonance

- Swinging without outside help free oscillations
- Swinging with outside help forced oscillations
- If  $\omega_d$  is a frequency of a driving force, then forced oscillations can be described by:

# Graphs

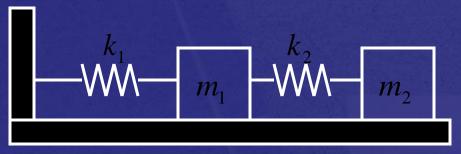
#### > The graphs show:

- (a) displacement as a function of time
- (b) velocity as a function of time
- (c) acceleration as a function of time
- The velocity is 90° out of phase with the displacement and the acceleration is 180° out of phase with the displacement



# Writing the Equations of Motion: Example 3

Write the equations of motion for the following:



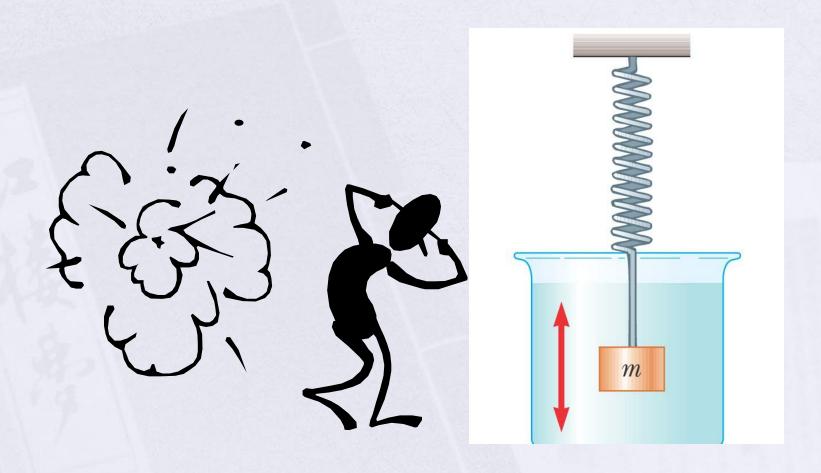
$$d_1$$
  $d_2$ 

$$0 = -m_1 \ddot{x}_1 - d_1 \dot{x}_1 - k_1 x_1 + k_2 (x_2 - x_1)$$

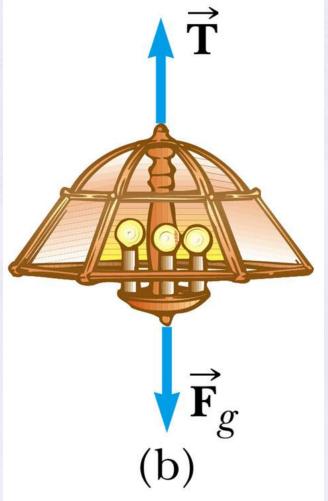
$$0 = -m_2 \ddot{x}_2 - d_2 \dot{x}_2 + k_2 (x_1 - x_2)$$



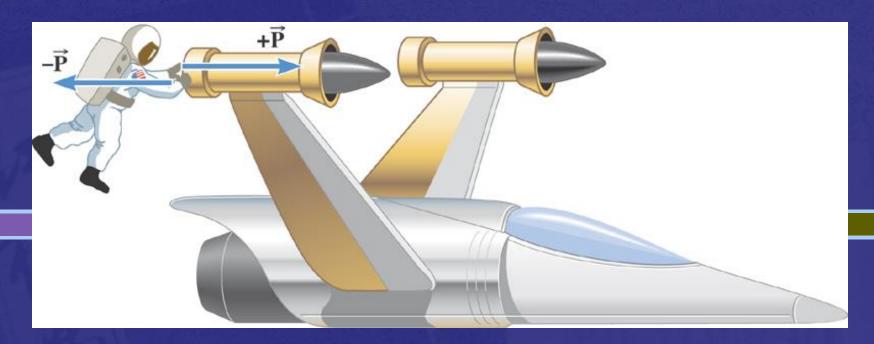
23







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$$x(t) = Ae^{i(\omega t - \varphi)}$$

$$\dot{x}(t) = i \omega A e^{i(\omega t - \varphi)} \quad \ddot{x}(t) = -\omega^2 A e^{i(\omega t - \varphi)}$$

$$-m\omega^{2}Ae^{i(\omega t-\varphi)}+ic\omega Ae^{i(\omega t-\varphi)}+kAe^{i(\omega t-\varphi)}=F_{0}e^{i\omega t}$$

$$-m\omega^2 A + ic\omega A + kA = F_0 e^{i\varphi} = F_0(\cos\varphi + i\sin\varphi)$$

$$A(k - m\omega^2) = F_0 \cos \varphi \qquad c\omega A = F_0 \sin \varphi$$

$$A^{2}(k - m\omega^{2})^{2} + c^{2}\omega^{2}A^{2} = F_{0}^{2}$$

SHM is the projection of uniform circular motion on a line in the plane of the motion. One period of SHM can be divided into 360°.

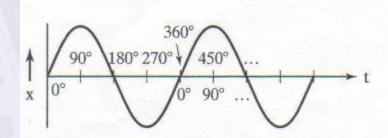


Figure 1-9 Phase of SHM curve.

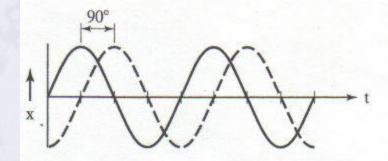


Figure 1-11 Solid curve 90° ahead of dashed curve.

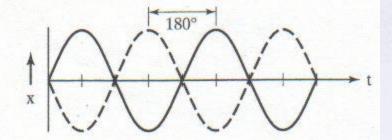


Figure 1-10 Two SHM curves differing in phase by 180° (out of phase).

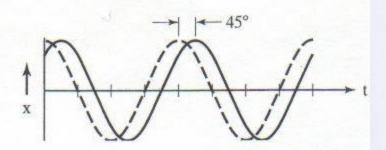


Figure 1-12 Solid curve 45° behind dashed curve.